



САМАРСКИЙ УНИВЕРСИТЕТ
SAMARA UNIVERSITY

Simplified thermodynamic model of gas turbine engine components

1 Constants

Table 1 — Constants for simplified thermodynamic calculation of a gas turbine engine

Symbol	Value	Unit of measure	Description
$c_{p.a}$	1,005	$\frac{\text{kJ}}{\text{kg}\cdot\text{K}}$	Isobaric heat capacity of air
k_a	1,4	-	Ratio of specific heats of air
R_a	0,287	$\frac{\text{kJ}}{\text{kg}\cdot\text{K}}$	Gas constant of air
$c_{p.cg}$	1,159	$\frac{\text{kJ}}{\text{kg}\cdot\text{K}}$	Isobaric heat capacity of combustion gas
k_{cg}	1,33	-	Ratio of specific heats of combustion gas
R_{cg}	0,2875	$\frac{\text{kJ}}{\text{kg}\cdot\text{K}}$	Gas constant of combustion gas
H_n	42 900	$\frac{\text{kJ}}{\text{kg}\cdot\text{K}}$	Net calorific value of kerosine

2 Atmosphere

Table 2 — Input data

Symbol	Unit of measure	Description
M_f	-	Flight Mach number
T_1	K	Ambient temperature
p_1	kPa	Ambient pressure

Ambient density:

$$\rho_1 = \frac{p_1}{R_a \cdot T_1}, \quad \frac{\text{kg}}{\text{m}^3}.$$

Flight velocity:

$$V_f = \sqrt{k_a \cdot 1000 \cdot R_a \cdot T_1} \cdot M_f, \quad \frac{\text{m}}{\text{s}};$$

$$V_{f.h} = 3.6 \cdot V_f, \quad \frac{\text{km}}{\text{h}}.$$

3 Air intake

Table 3 — Input data

Symbol	Unit of measure	Description
M_f	-	Flight Mach number
T_1	K	Ambient temperature
p_1	kPa	Ambient pressure
σ_{intake}	-	Intake total pressure loss

Total temperature of atmospheric airflow:

$$T_1^* = T_1 \cdot \left(1 + \frac{k_a - 1}{2} \cdot M_f^2\right), \text{ K} .$$

Total pressure of atmospheric airflow:

$$p_1^* = p_1 \cdot \left(1 + \frac{k_a - 1}{2} \cdot M_f^2\right)^{\frac{k_a}{k_a - 1}}, \text{ kPa} .$$

Total temperature at discharge:

$$T_2^* = T_1^*, \text{ K} .$$

Total pressure at discharge:

$$p_2^* = p_1^* \cdot \sigma_{\text{intake}}, \text{ kPa} .$$

4 Compressor

Table 4 — Input data

Symbol	Unit of measure	Description
T_2^*	K	Total temperature at entrance
p_2^*	kPa	Total pressure at entrance
π_c^*	-	Compressor pressure ratio
η_c^*	-	Compressor efficiency

Compressor work:

$$L_c = c_{pa} \cdot T_2^* \cdot \left(\pi_c^{*\frac{k_a-1}{k_a}} - 1 \right) \cdot \frac{1}{\eta_c^*} , \quad \frac{\text{kJ}}{\text{kg}} .$$

Total temperature at discharge:

$$T_3^* = T_2^* + \frac{L_c}{c_{pa}} , \quad \text{K} .$$

Total pressure at discharge:

$$p_3^* = p_2^* \cdot \pi_c^* , \quad \text{kPa} .$$

5 Fan

Table 5 — Input data

Symbol	Unit of measure	Description
T_2^*	K	Total temperature at entrance
p_2^*	kPa	Total pressure at entrance
m	-	Bypass ratio
π_{Ipc}^*	-	Fan pressure ratio at primary flowpath
η_{Ipc}^*	-	Fan efficiency at primary flowpath
π_{fII}^*	-	Fan pressure ratio at secondary flowpath
η_{fII}^*	-	Fan efficiency at secondary flowpath

Fan work at primary flowpath:

$$L_{\text{Ipc}} = c_{pa} \cdot T_2^* \cdot (\pi_{\text{Ipc}}^{*\frac{k_a-1}{k_a}} - 1) \cdot \frac{1}{\eta_{\text{Ipc}}^*}, \quad \frac{\text{kJ}}{\text{kg}}.$$

Total temperature at primary flowpath discharge:

$$T_3^* = T_2^* + \frac{L_{\text{Ipc}}}{c_{pa}}, \quad \text{K}.$$

Total pressure at primary flowpath discharge:

$$p_3^* = p_2^* \cdot \pi_{\text{Ipc}}^*, \quad \text{kPa}.$$

Fan work at secondary flowpath:

$$L_{\text{fII}} = c_{pa} \cdot T_2^* \cdot (\pi_{\text{fII}}^{*\frac{k_a-1}{k_a}} - 1) \cdot \frac{1}{\eta_{\text{fII}}^*}, \quad \frac{\text{kJ}}{\text{kg}}.$$

Total temperature at secondary flowpath discharge:

$$T_{3-\text{II}}^* = T_2^* + \frac{L_{\text{fII}}}{c_{pa}}, \quad \text{K}.$$

Total pressure at secondary flowpath discharge:

$$p_{3-\text{II}}^* = p_2^* \cdot \pi_{\text{fII}}^*, \quad \text{kPa}.$$

6 Combustion chamber

Table 6 — Input data

Symbol	Unit of measure	Description
T_3^*	K	Total temperature at entrance
p_3^*	kPa	Total pressure at entrance
$g_{\text{cooling.sb.hpt}}$	-	Relative mass flow rate of cooling air for stationary blades of high pressure turbine
$g_{\text{cooling.rb.hpt}}$	-	Relative mass flow rate of cooling air for rotational blades of high pressure turbine
$g_{\text{cooling.sb.ipt}}$	-	Relative mass flow rate of cooling air for stationary blades of intermediate pressure turbine
$g_{\text{cooling.rb.ipt}}$	-	Relative mass flow rate of cooling air for rotational blades of intermediate pressure turbine
$g_{\text{cooling.sb.lpt}}$	-	Relative mass flow rate of cooling air for stationary blades of low pressure turbine
$g_{\text{cooling.rb.lpt}}$	-	Relative mass flow rate of cooling air for rotational blades of low pressure turbine
σ_{cc}	-	Combustion chamber total pressure loss
η_{cc}	-	Combustion efficiency
T_4^*	K	Total temperature at discharge

Relative fuel consumption:

$$q_f = \frac{c_{p.cg} \cdot T_4^* - c_{p.a} \cdot T_3^*}{H_n \cdot \eta_{\text{cc}}} .$$

Total pressure at discharge:

$$p_4^* = p_3^* \cdot \sigma_{\text{cc}} , \quad \text{kPa} .$$

Relative mass flow rate at discharge:

$$v_4 = \left(1 - \sum_i g_{\text{cooling.i}} \right) \cdot (1 + q_f) .$$

7 Turbine (compressor driving)

Table 7 — Input data

Symbol	Unit of measure	Description
T_4^*	K	Total temperature at entrance
p_4^*	kPa	Total pressure at entrance
v_4	-	Relative mass flow rate at entrance
T_3^*	K	Total temperature of cooling air (at compressor discharge)
$g_{\text{cooling.sb.t}}$	-	Relative mass flow rate of cooling air for stationary blades of the turbine
$g_{\text{cooling.rb.t}}$	-	Relative mass flow rate of cooling air for rotational blades of the turbine
L_c	$\frac{\text{kJ}}{\text{kg}}$	Compressor work
η_m	-	Mechanical loss
η_t^*	-	Turbine efficiency

Total temperature at discharge of the first stationary blade row:

$$T_{\text{sb.t}}^* = \frac{c_{p.cg} \cdot T_4^* \cdot v_4 + c_{p.a} \cdot T_3^* \cdot g_{\text{cooling.sb.t}}}{c_{p.cg} \cdot (v_4 + g_{\text{cooling.sb.t}})}, \quad \text{K} .$$

Turbine work:

$$L_t = \frac{L_c}{\eta_m \cdot (v_4 + g_{\text{cooling.sb.t}})}, \quad \frac{\text{kJ}}{\text{kg}} .$$

Turbine pressure ratio:

$$\pi_t^* = \left(1 - \frac{L_t}{c_{p.cg} \cdot T_{\text{sb.t}}^* \cdot \eta_t^*}\right)^{-\frac{k_{cg}}{k_{cg}-1}} .$$

Total temperature at discharge of the last rotational blade row:

$$T_{\text{rb.t}}^* = T_{\text{sb.t}}^* - \frac{L_t}{c_{p.cg}}, \quad \text{K} .$$

Total temperature at discharge:

$$T_5^* = \frac{c_{p.cg} \cdot T_{\text{rb.t}}^* \cdot (v_4 + g_{\text{cooling.sb.t}}) + c_{p.a} \cdot T_3^* \cdot g_{\text{cooling.rb.t}}}{c_{p.cg} \cdot (v_4 + g_{\text{cooling.sb.t}} + g_{\text{cooling.rb.t}})}, \quad \text{K} .$$

Total pressure at discharge:

$$p_5^* = \frac{p_4^*}{\pi_t^*}, \text{ kPa} .$$

Relative mass flow rate at discharge:

$$v_5 = v_4 + g_{\text{cooling.sb.t}} + g_{\text{cooling.rb.t}} .$$

8 Turbine (fan driving)

Table 8 — Input data

Symbol	Unit of measure	Description
T_4^*	K	Total temperature at entrance
p_4^*	kPa	Total pressure at entrance
v_4	-	Relative mass flow rate at entrance
T_3^*	K	Total temperature of cooling air (at compressor discharge)
$g_{\text{cooling .sb. lpt}}$	-	Relative mass flow rate of cooling air for stationary blades of the turbine
$g_{\text{cooling .rb. lpt}}$	-	Relative mass flow rate of cooling air for rotational blades of the turbine
m	-	Bypass ratio
L_{lpc}	$\frac{\text{kJ}}{\text{kg}}$	Fan work at primary flow
L_{fII}	$\frac{\text{kJ}}{\text{kg}}$	Fan work at secondary flow
$\eta_{m.\text{lp}}$	-	Mechanical loss
η_{lpt}^*	-	Turbine efficiency

Total temperature at discharge of the first stationary blade row:

$$T_{\text{sb.lpt}}^* = \frac{c_{p.cg} \cdot T_4^* \cdot v_4 + c_{p.a} \cdot T_3^* \cdot g_{\text{cooling.sb.lpt}}}{c_{p.cg} \cdot (v_4 + g_{\text{cooling.sb.lpt}})}, \quad \text{K} .$$

Turbine work:

$$L_{\text{lpt}} = \frac{L_{\text{lpc}} + m \cdot L_{\text{fII}}}{\eta_{m.\text{lp}} \cdot (v_4 + g_{\text{cooling.sb.lpt}})}, \quad \frac{\text{kJ}}{\text{kg}} .$$

Turbine pressure ratio:

$$\pi_{\text{lpt}}^* = \left(1 - \frac{L_{\text{lpt}}}{c_{p.cg} \cdot T_{\text{sb.lpt}}^* \cdot \eta_{\text{lpt}}^*} \right)^{-\frac{k_{cg}}{k_{cg}-1}} .$$

Total temperature at discharge of the last rotational blade row:

$$T_{\text{rb.lpt}}^* = T_{\text{sb.lpt}}^* - \frac{L_{\text{lpt}}}{c_{p.cg}}, \quad \text{K} .$$

Total temperature at discharge:

$$T_5^* = \frac{c_{p.cg} \cdot T_{rb.lpt}^* \cdot (v_4 + g_{cooling.sb.lpt}) + c_{p.a} \cdot T_3^* \cdot g_{cooling.rb.lpt}}{c_{p.cg} \cdot (v_4 + g_{cooling.sb.lpt} + g_{cooling.rb.lpt})} , \text{ K} .$$

Total pressure at discharge:

$$p_5^* = \frac{p_4^*}{\pi_{lpt}^*} , \text{ kPa} .$$

Relative mass flow rate at discharge:

$$v_5 = v_4 + g_{cooling.sb.lpt} + g_{cooling.rb.lpt} .$$

9 Bypass duct

Table 9 — Input data

Symbol	Unit of measure	Description
T_{3-II}^*	K	Total temperature at entrance
p_{3-II}^*	kPa	Total pressure at entrance
σ_{II}	-	Duct total pressure loss

Total temperature at discharge:

$$T_{5-II}^* = T_{3-II}^* , \text{ K} .$$

Total pressure at discharge:

$$p_{5-II}^* = p_{3-II}^* \cdot \sigma_{II} , \text{ kPa} .$$

10 Mixer

Table 10 — Input data

Symbol	Unit of measure	Description
T_5^*	K	Total temperature at primary flow entrance
p_5^*	kPa	Total pressure at primary flow entrance
v_5	-	Relative mass flow rate at primary flow entrance
T_{5-II}^*	K	Total temperature at secondary flow entrance
p_{5-II}^*	kPa	Total pressure at secondary flow entrance
m	-	Bypass ratio
σ_m	-	Mixer total pressure loss

Total temperature at discharge:

$$T_6^* = \frac{c_{p.cg} \cdot T_5^* \cdot v_5 + c_{p.a} \cdot T_{5-II}^* \cdot m}{c_{p.cg} \cdot (v_5 + m)}, \quad \text{K} .$$

Total pressure at discharge:

$$p_6^* = \frac{p_5^* \cdot v_5 + p_{5-II}^* \cdot m}{v_5 + m} \cdot \sigma_m, \quad \text{kPa} .$$

Relative mass flow rate at discharge:

$$v_6 = \frac{v_5 + m}{1 + m} .$$

11 Exhaust nozzle (primary)

Table 11 — Input data

Symbol	Unit of measure	Description
T_8^*	K	Total temperature at entrance
p_8^*	kPa	Total pressure at entrance
v_8	-	Relative mass flow rate at entrance
p_1	kPa	Ambient pressure
φ_n	-	Nozzle velocity loss

Nozzle pressure ratio at complete expansion:

$$\pi_{n.c} = \frac{p_8^*}{p_1} .$$

Exit velocity:

$$c_9 = \varphi_n \sqrt{2000 \cdot c_{p.cg} \cdot T_8^* \cdot \left(1 - \pi_{n.c}^{\frac{k_{cg}-1}{k_{cg}}}\right)} , \quad \frac{\text{m}}{\text{s}} .$$

Static temperature at discharge:

$$T_9 = T_8^* - \frac{c_9^2}{2000 \cdot c_{p.cg}} , \quad \text{K} .$$

Relative mass flow rate at discharge:

$$v_9 = v_8 .$$

12 Exhaust nozzle (secondary)

Table 12 — Input data

Symbol	Unit of measure	Description
T_{8-II}^*	K	Total temperature at entrance
p_{8-II}^*	kPa	Total pressure at entrance
p_1	kPa	Ambient pressure
φ_{nII}	-	Nozzle velocity loss

Nozzle pressure ratio at complete expansion:

$$\pi_{nII.c} = \frac{p_{8-II}^*}{p_1} .$$

Exit velocity:

$$c_{9-II} = \varphi_{nII} \sqrt{2000 \cdot c_{p.a} \cdot T_{8-II}^* \cdot \left(1 - \pi_{nII.c}^{\frac{k_a-1}{k_a}}\right)} , \quad \frac{m}{s} .$$

Static temperature at discharge:

$$T_{9-II} = T_{8-II}^* - \frac{c_{9-II}^2}{2000 \cdot c_{p.a}} , \quad K .$$

13 Features of a turbojet

Table 13 — Input data

Symbol	Unit of measure	Description
V_f	$\frac{m}{s}$	Flight velocity
c_9	$\frac{m}{s}$	Exit velocity
v_9	-	Relative mass flow rate at discharge of exhaust nozzle
q_f	-	Relative fuel consumption
v_4	-	Relative mass flow rate at discharge of the combustion chamber
P	kN	Thrust

Specific thrust:

$$S_p = 0,001 \cdot (c_9 \cdot v_9 - V_f) , \quad \frac{\text{kN} \cdot \text{s}}{\text{kg}} .$$

Total air mass flow rate throw the engine:

$$G_2 = \frac{P}{S_p} , \quad \frac{\text{kg}}{\text{s}} .$$

Fuel consumption:

$$G_f = 3600 \cdot q_f \cdot \frac{v_4}{1 + q_f} \cdot G_2 , \quad \frac{\text{kg}}{\text{h}} .$$

Specific fuel consumption:

$$S_{Gf} = \frac{G_f}{P} , \quad \frac{\text{kg}}{\text{kN} \cdot \text{h}} .$$

14 Features of a turbofan

Table 14 — Input data

Symbol	Unit of measure	Description
V_f	$\frac{m}{s}$	Flight velocity
c_9	$\frac{m}{s}$	Exit velocity at primary exhaust nozzle
v_9	-	Relative mass flow rate at discharge of primary exhaust nozzle
c_{9-II}	$\frac{m}{s}$	Exit velocity at secondary exhaust nozzle
m	-	Bypass ratio
q_f	-	Relative fuel consumption
v_4	-	Relative mass flow rate at discharge of the combustion chamber
P	kN	Thrust

Specific thrust:

$$S_p = 0,001 \frac{(c_9 \cdot v_9 - V_f) + m \cdot (c_{9-II} - V_f)}{1 + m}, \quad \frac{\text{kN} \cdot \text{s}}{\text{kg}} .$$

Total air mass flow rate throw the engine:

$$G_2 = \frac{P}{S_p}, \quad \frac{\text{kg}}{\text{s}} .$$

Air mass flow rate throw the primary flowpath:

$$G_{2-I} = \frac{G_2}{1 + m}, \quad \frac{\text{kg}}{\text{s}} .$$

Air mass flow rate throw the secondary flowpath:

$$G_{2-II} = m \cdot G_{2-I}, \quad \frac{\text{kg}}{\text{s}} .$$

Fuel consumption:

$$G_f = 3600 \cdot q_f \cdot \frac{v_4}{1 + q_f} \cdot G_{2-I}, \quad \frac{\text{kg}}{\text{h}} .$$

Specific fuel consumption:

$$S_{Gf} = \frac{G_f}{P}, \quad \frac{\text{kg}}{\text{kN} \cdot \text{h}} .$$

15 Features of a turbofan with mixing

Table 15 — Input data

Symbol	Unit of measure	Description
V_f	$\frac{m}{s}$	Flight velocity
c_9	$\frac{m}{s}$	Exit velocity at primary exhaust nozzle
v_9	-	Relative mass flow rate at discharge of primary exhaust nozzle
m	-	Bypass ratio
q_f	-	Relative fuel consumption
v_4	-	Relative mass flow rate at discharge of the combustion chamber
P	kN	Thrust

Specific thrust:

$$S_p = 0,001 \cdot (c_9 \cdot v_9 - V_f) , \quad \frac{\text{kN} \cdot \text{s}}{\text{kg}} .$$

Total air mass flow rate throw the engine:

$$G_2 = \frac{P}{S_p} , \quad \frac{\text{kg}}{\text{s}} .$$

Air mass flow rate throw the primary flowpath:

$$G_{2-I} = \frac{G_2}{1+m} , \quad \frac{\text{kg}}{\text{s}} .$$

Air mass flow rate throw the secondary flowpath:

$$G_{2-II} = m \cdot G_{2-I} , \quad \frac{\text{kg}}{\text{s}} .$$

Fuel consumption:

$$G_f = 3600 \cdot q_f \cdot \frac{v_4}{1+q_f} \cdot G_{2-I} , \quad \frac{\text{kg}}{\text{h}} .$$

Specific fuel consumption:

$$S_{Gf} = \frac{G_f}{P} , \quad \frac{\text{kg}}{\text{kN} \cdot \text{h}} .$$

